



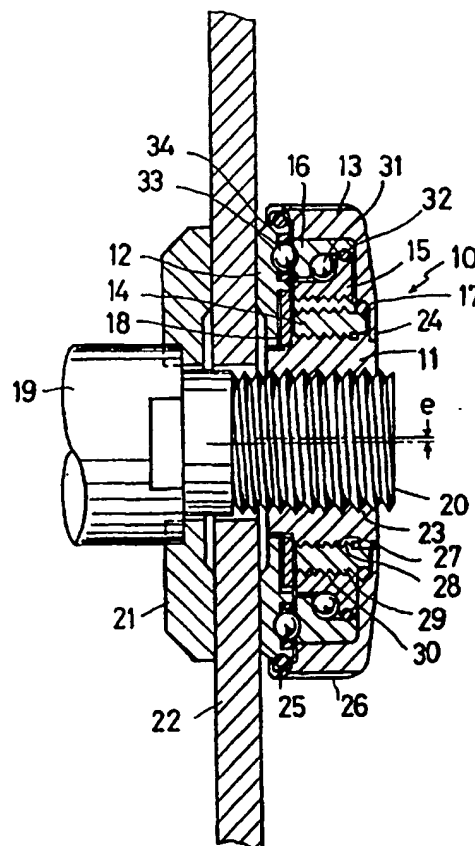
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(54) Title: SCREW FOR FASTENING

(57) Abstract

The present invention aims to provide a screw (10) for fastening which is capable of being both fastened and loosened strongly simply with a direct manual rotating operation. The device (10) is provided with a screw member (11) with a female screw or a male screw formed in its core area; a flange ring (12) which is fitted around the outer circumferential area of one edge of said screw member the frontal surface of which makes contact with the object to be attached; an operating ring (13) which faces the flange ring (12), separated from it by a prescribed separation, and which inputs the operational rotational force; a reverse screw ring (14) which is positioned between the flange ring (12) and the operating ring (13), and is screwed onto the outer circumferential area of the screw member (11), regulating the screw feed amount; a screw feed ring (15) which is off-centred by screwing part of the female screw area provided on the relevant inner circumferential portion onto a male screw area provided on the outer circumference of the reverse screw ring (14), and which transmits the screw feed to the above-mentioned flange ring (12); and a holding ring (16) which permits revolution of the said screw feed ring (15) by holding the said screw feed ring (15) off-center.



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SCREW FOR FASTENING**Detailed Description of the Invention****Field of Industrial Use**

The present invention relates to a screw for fastening such as an attachment bolt or an attachment nut used to secure a rotational tool, such as the grindstone of a hand grinder or a circular hand saw, for example, to the drive shaft of an electrically powered tool.

Prior Art

Conventionally, in order to attach a rotational tool such as the grindstone or circular saw mentioned above to the drive shaft of an electrically powered tool, a flange and bolt are formed on the end of the drive shaft, the rotational tool is impaled on the bolt portion and a nut screwed onto the outside thereof, and the nut is fastened to secure the rotational tool between the said nut and the abovementioned flange.

However, there are problems in using the abovementioned rotational tool in that, if the said rotational tool is used in an impacting fashion, the nut becomes tighter than necessary due to the impact and the reaction thereto, so that the nut is too tight when the rotational tool is to be removed and it cannot be removed even if a wrench is used.

Additionally, the abovementioned operations of fastening and loosening the nut have problems in that normal practice is to employ a force-enhancing tool such as a wrench, but the force with which the nut is fastened or loosened is amplified only by the force-enhancing tool and adequate amplification can therefore not be achieved.

Problems To Be Solved By The Invention

The present invention aims to provide a screw for fastening whereby the screw member can be readily removed by preventing the screw member from becoming too tight due to the rotational force caused by reaction to use of the rotational tool in an impacting fashion.

Additionally, it aims to provide a screw for fastening which gives a substantial fastening force and a strong loosening force for a minor rotational input, and which can be strongly fastened and loosened by turning it in a direct manual rotating operation without using a power-enhancing tool.

Again, it aims to provide a screw for fastening which can be compactly constructed and has good operating properties.

Means Of Resolving the Problems

The invention disclosed in claim 1 of the invention is characterised in that it is a screw for fastening which is provided with a screw member with a female screw or a male screw formed in its core area; a flange ring which is fitted around the outer circumferential area of one edge of said screw member with freedom to turn relative to said screw member, and the outside surface of which makes contact with the object to be attached; an operating ring which is fitted around the outer circumferential area at the other edge of the abovementioned screw member with freedom to turn relative to said screw member, which operating ring faces the abovementioned flange ring, separated from it by a prescribed separation, and which inputs the operational rotational force; a screw feed ring which is positioned in the interval between the abovementioned flange ring and the abovementioned operating ring, which is held off-center by having part of its female screw area,

which is provided on the relevant inner circumferential portion, mesh with the male screw area provided on the outer circumferential area of the screw member in the said positioning interval, and which transmits screw feed caused by revolution to the abovementioned flange ring in the thrust direction; and a holding ring fitted around the outer circumferential area of the abovementioned screw feed ring, which permits revolution of the said screw feed ring by holding the said screw feed ring off-center.

The invention claimed in claim 2 of the invention is characterised in that it is a screw for fastening which, in addition to having the configuration of the invention claimed in claim 1 mentioned above, is a screw for fastening in which a reverse screw ring regulated to a predetermined screw feed amount is screwed onto the outer circumference of the abovementioned screw member; the outer circumference of the said reverse screw ring is provided with a male screw area meshing with part of the female screw area of the abovementioned screw feed ring; and in which a rotation transmitting member, which transmits the rotation of the operating ring to the reverse screw ring and which slips at more than a predetermined load, is inserted in the region where the outer circumferential area of the other edge of the abovementioned reverse screw ring and the inner circumferential area of the abovementioned operating ring face each other.

The invention claimed in claim 3 of the invention is characterised in that it is a screw for fastening which, in addition to the configuration of the invention claimed in claim 1 mentioned above, is a screw for fastening in which a torque limiter, which cuts off the rotational

force from the flange ring at a predetermined load, is inserted between the facing surfaces of the said flange ring and screw member.

As regards the invention disclosed in claim 1
5 of the invention, a rotational force is input by fitting the screw of the screw member to the screw area on the side where the object to be attached is and turning the operating ring in the fastening direction, whereupon the screw member also turns in
10 the fastening direction under the frictional resistance of the internal structural elements and is screwed onto the screw area where the object to be attached is.

Once the outside surface of the flange ring
15 has come into contact with the object to be attached due to this screw fitting action, the contact load is applied via the screw feed ring to the screw member, so that the screw member is prevented from turning and only the turning operation of the
20 operating ring is permitted.

If the operating ring in this state experiences a further rotational force input in the fastening direction, the screw feed ring is supported via the holding ring in a state off-
25 center of the operating ring and, therefore, the screw feed ring revolves under the rotation of the operating ring, and part of the female screw area of the screw feed ring is screwed onto the male screw area on the screw member side and, therefore,
30 the screw feed ring is screw fed to the flange ring side accompanying the revolution mentioned above, and this screw feeding action is transmitted to the flange ring and the relevant surface of the said flange ring is pressured, under the force in the
35 thrust direction, into contact with the object to be attached, which is thereby secured. In other words, without turning the screw member, the object

to be attached is pressed into contact and secured via the flange ring under the force of the screw feed ring in the thrust direction. Moreover, the screw feed of the screw feed ring discussed above is stepped down by

$$P (1-D1/D2)$$

where, P = the screw pitch of the female screw area of the screw feed ring, and the male screw area on the screw member side which screws onto the said female screw area

$D1$ = the independent effective diameter of the male screw area on the screw member side

$D2$ = the independent effective diameter of the female screw area of the screw feed ring,

and, therefore, the torque of the rotational force of the operating ring is increased by the amount of step-down, and the flange ring is pressed into contact under a substantial force in the thrust direction, and the object to be attached can be strongly fastened.

Moreover, when the attached object is to be removed, the operating ring is rotated in the opposite direction to that mentioned above (the unfastening direction). Because there is a high pressure-contact load from the flange ring on the object to be attached in the initial stages of the rotation, then, if the operating ring is rotated in the opposite direction, the screw feed ring can rotate backwards under a rotational force of increased torque in the same way as described above, the said screw feed ring can be returned to the initial position prior to screw feeding, and the pressure contacting of the flange ring on the

object to be fitted is released by this returning action, and, therefore, the load from the flange ring side of the screw member is released, the rotation of the operating ring is transmitted to the screw member by the frictional resistance of the internal structural elements, and the screw member can be made to rotate in reverse and can be removed by turning the operating ring.

In the invention described in claim 2 of the invention, in addition to the actin of the invention described in claim 1 mentioned above, rotating the operating ring in the fastening direction prevents the screw member from rotating and, if the operating ring experiences a rotational force input in the fastening direction, the reverse screw ring first experiences a rotational force from the operating ring via the rotational-transmitting member and is screw fed to the fastening side as far as the regulated position, the reverse screw ring is regulated and stops rotating, and then the rotation-transmitting member slips with respect to the operating ring so that the rotational force is cut off, and this means that the screw feed ring revolves and is screw fed as described above so that the object to be attached is pressed into contact and secured in the thrust direction, via the flange ring, due to the screw feeding action of the said screw feeding ring.

The object to be attached is removed by the reverse rotation of the operating ring: until the pressure contact of the flange ring loosens, the operating ring and the rotation-transmitting member slip, the screw feed ring revolves and withdraws and then the pressure contact of the flange ring is released, whereupon the rotation of the operating ring is transmitted via the rotation-transmitting

member to the reverse screw ring which is withdrawn, and the abovementioned screw feed ring and reverse screw ring can be returned to the initial position by having this withdrawing action proceed as far as the regulating position.

In the invention described in claim 3 of the invention, in addition to the action of the invention described in claim 1 mentioned above, once an impact load from the object to be attached is applied to the flange ring, the said impact load is cut off by the torque limiter once it has exceeded the setting load of the torque limiter, thereby permitting the flange ring to rotate and making it possible to avoid the impacts mentioned above, and, moreover, making it possible to prevent the said impact load from acting on the screw feed ring, reverse screw ring, screw member, holding ring, operating ring and the like.

Using the present invention, the object to be attached is pressed into contact and secured under the force of the flange ring in the thrust direction and this pressurised contact has no energy component in the direction of rotation, which means that the screw member can be reliably prevented from becoming too tight due to rotational forces caused by reaction to the use of the said rotational tool in an impacting fashion (when, by way of example, the object to be attached is a rotational tool), and the screw member can be readily removed. Moreover, the impact load of the rotational tool mentioned above is cut off by the torque limiter, thereby making it possible to prevent damage to the rotational tool from the impact load, and also to prevent damage to the internal parts of the screw for fastening.

In addition, the screw feed ring presses the flange ring under a force of increased torque and

this gives a substantial fastening force and strong loosening force on the flange ring for a minor rotational input at the operating ring, and it is also possible to fasten and loosen the device strongly by a direct manual rotating operation without using a power-enhancing tool.

In addition, by inserting a reverse screw ring between the screw member and the screw feed ring, the screw feed ring is more reliably returned to the initial position and a good operating mode is obtained.

In addition, housing structural elements such as the flange ring, operating ring, screw feed ring, reverse screw ring and the holding ring in the area around the screw member gives a screw for fastening which is compactly constructed and has a good operating performance.

Brief Description of the Diagrams

Figure 1 is a cross-sectional diagram of a screw for fastening.

Figure 2 is an exploded cross-sectional diagram of a screw for fastening.

Figure 3 is a side elevation of a torque limiter.

Figure 4 is a cross-sectional diagram showing another example of a screw for tightening.

Detailed Description of Preferred Embodiments

An embodiment of the invention is now described based on the following figures.

The figures show a screw for fastening which has been formed in a nut shape: in Figure 1 and Figure 2, the said screw 10 for fastening comprises a screw member 11, a flange ring 12, an operating ring 13, a reverse screw ring 14, a screw-feed ring 15, a holding ring 16, an O-ring 17 which is a rotation-transmitting member, and a torque limiter 18; and, by way of example, the screw 10 for

fastening secures the rotational tool 22, which is the object to be attached, for example the grindstone of a hand grinder, between itself and an inner flange 21 attached to the drive shaft 19 of an electrically powered tool by screwing onto the attachment bolt 20 of the said drive shaft 19.

As regards the screw member 11 of the screw 10 for fastening mentioned above, it is provided on the inner circumference of its shaft core portion with a female screw 23 which screws onto the attachment bolt 20 of the abovementioned drive shaft 19, it has stepped areas formed on both the inside and outside edges of its outer circumferential area, it has a flange ring 12 rotatably fitted around its inside edge, and it has the regulating area 24 of the operating ring 13 rotatably fitted around its outside edge side.

The inside of the abovementioned operating ring 13 is formed in a recessed shape, thereby forming a space within it, and the inside of its outside edge overlaps with the outer circumferential area of the abovementioned flange ring 12, and an O-ring 25 is inserted to seal the said overlapping portion. Further, a knurled rim 26 is formed at the outer circumferential area of the abovementioned operating ring 13, thereby facilitating manual rotational input.

A male screw area 27 is provided on the outer circumferential area of the abovementioned screw member 11 in the central area sandwiched between the stepped areas at both edges, and a female screw area 28 provided on the inner circumference of the abovementioned reverse screw ring 14 is screwed onto this male screw area 27 so that, when a rotational force is applied to this reverse screw ring 14, the reverse screw ring 14 is screw fed slightly forwards or backwards within a range

regulated by the regulating area 24 of the operating ring 13 and the torque limiter 18, for example a range of about 1 mm.

Further the abovementioned O-ring 17 is inserted in the region where the outer circumference on the outside edge of the abovementioned reverse screw ring 14 and the inner circumferential area of the abovementioned operating ring 13 face each other, this O-ring 17 being provided not only for its sealing action, but also to rotate the reverse screw ring 14 by transmitting the rotational force of the operating ring 13 by frictional resistance, and also to slip the transmission when the return screw ring 14 experiences a load from the flange ring 12 side.

A male screw area 29 is provided on the outer circumferential area of the abovementioned return screw ring 14, while a female screw area 30 with a screw diameter slightly larger than that of, and a screw pitch the same as that of the abovementioned male screw area 29 is provided on the inner circumferential surface of the abovementioned screw feed ring 15, and the screw feed ring 15 is off-centred by an off-centring amount e such that part of the said female screw area 30 meshes with the above male screw area 29, and the holding ring 16 holds this off-center position of the screw feed ring 15 via bearings 31. Moreover, the holding ring 16 is secured by insertion under pressure against the recessed inner surface of the operating ring 13. Further, an O-ring 32 is inserted between the abovementioned screw feed ring 15 and the holding ring 16.

There is play between the operating ring 13 and the flange ring 12 such that the whole of the operating ring 13 can move to the flange ring 12

when the screw feed ring 15 is screw fed by a prescribed amount, for example about 1 mm.

Thrust bearings 33 are inserted between the facing surfaces of the abovementioned holding ring 16 and the abovementioned flange ring 12, and the said thrust bearings 33 are held in a retainer 34 fitted into a recess in the flange ring 12.

As shown in Figure 3, the abovementioned torque limiter 18 comprises a resilient ring 35, and a plurality of recessed areas 37 in the flange ring 12 which engage with a plurality of claws 36 formed in this resilient ring 35. Spline channels 38 are provided in the inner circumferential area of the abovementioned resilient ring 35, and the spline channels 38 engage with spline claws 39 formed in the recesses of the stepped area on the inside of the screw member 11 with which they turn in an integral fashion. Further, the plurality of claws 36 ... are formed at the ends of arms 40 which resiliently alter position in the inward and outward direction, and the arms 40 will alter position and slip to cut off any driving forces if a rotational force larger than the resiliency set in the arms 40 is transmitted from the flange ring 12 to the resilient ring 35.

The operation of a screw 10 for fastening which has been constructed in this manner is described below.

In attaching the abovementioned rotational tool 22 to the attachment bolt 20 of the drive shaft 19 using the abovementioned screw 10 for fastening, the flange ring 12 side of the screw 10 for fastening is made to face the side with the rotational tool 22, the female screw 23 of the screw member 11 of the screw 10 for fastening is fitted to the attachment bolt 20, and the knurled rim 26 portion of the operating ring 13 is turned

directly by hand in the fastening direction,
thereby screwing on the female screw 23.

When the flange ring 12 is not in contact with
the rotational tool 22 during this screwing-on
5 action, the rotation of the operating ring 13 is
transmitted to the screw member 11 by the internal
resistance of the various elements formed on the
inside of the operating ring 12, for example by the
frictional resistance of such elements as the
10 reverse screw ring 14, screw feed ring 15, holding
ring 16 and torque limiter 18, and the said screw
member 11 turns in the fastening direction and is
screwed onto the attachment bolt 20.

Once the outside face of the flange ring 12
15 has come into contact with the side surface of the
rotational tool 22 due to this screwing-on action,
the contacting load acts on the screw member 11 via
the thrust bearings 33, holding ring 16, bearings
31, screw feed ring 15, return screw ring 14 or
20 torque limiter 18, and the screw member 11 is
prevented from turning.

If, in this state, the operating ring 13 is
again rotated in the fastening direction, the
return screw ring 14 is screw fed by the meshing of
25 the male and female screw areas 27 and 28 under the
frictional resistance of the O-ring 17 (a feed of
about 1 mm), and the inside edge comes into contact
with the resilient ring 35 of the torque limiter 18
and is regulated, slip occurs between the O-ring 17
30 and the return screw ring 14 and only the turning
operation of the operating ring 13 is permitted.

If, in this state, the operating ring 13 is
again rotated in the fastening direction, the screw
feed ring 15 is supported, via the holding ring 16,
35 in a state off-centred from the operating ring 13,
and, therefore, the screw feed ring 15 revolves
under the rotation of the operating ring 13, and,

because part of the female screw area 30 of the screw feed ring 15 is meshed with the male screw area 29 of the reverse screw ring 14, the screw feed ring 15 is screw fed (feed of about 1 mm) to the flange ring 12 side, and this screw feeding is transmitted by the bearing 31, the holding ring 16 and the thrust bearings 33 to the flange ring 12, and the said side face of the said flange ring 12 is pressured into contact with the rotational tool 22 and secures it under the force in the thrust direction.

Moreover, the screw feed of the abovementioned screw feed ring 15 is stepped down by

$P (1-D1/D2)$

where, P = the screw pitch of the female screw area 30 of the screw feed ring 15, and

the male screw area 29 of the reverse screw ring 14

$D1$ = the independent effective diameter of the male screw area 29 of the reverse screw ring 14

$D2$ = the independent effective diameter of the female screw area 29 of the screw feed ring 15,

and, therefore, the torque of the rotational force of the operating ring 13 is increased by the amount of step-down, and the flange ring 12 is pressed into contact under a substantial force, and the rotational tool 22 can be strongly fastened in the thrust direction.

As regards the use of a rotational tool 22 attached as described above, if the flange ring 12 experiences an impact load due to the use of the said rotational tool 22 in an impacting fashion, and if the said impact load is greater than the set

load of the torque limiter 18, which is to say the resilience set in the arms 40 of the resilient ring 35, then the claws 36 come away from the recessed areas 37, the said impact load is cut off by the torque limiter 18 and the flange ring 12 is permitted to rotate so that the abovementioned impact can be avoided, and the said impact load can be prevented from acting on the holding ring 16, bearings 31, screw feed ring 15, reverse screw ring 14 and screw member 11.

Moreover, when the rotational tool 22 is to be removed, the operating ring 13 is rotated in the opposite direction to that mentioned above (the unfastening direction). Because there is a high pressure-contact load from the flange ring 12 on the rotational tool 22 in the initial stages of the rotation, if the operating ring 13 is rotated in the opposite direction, the operating ring 13 and the O-ring 17, which is the rotation-transmitting member, slip and the screw feed ring 15 revolves and withdraws until the pressure contact of the flange ring 12 is relaxed.

Once the pressure contact of the flange ring 12 has then been released, the rotation of the operating ring 13 causes the reverse screw ring 14 to withdraw via the O-ring 17 and, if this withdrawal proceeds to the position regulated by the regulating area 24 of the operating ring 13, then the abovementioned screw feed ring 15 and the reverse screw ring 14 can be returned to the initial position.

If, once they have been returned to the initial position as described above, the operating ring 13 is again turned in the removal direction, then the whole unit turns under internal resistance as described above and the screw member 11 is made

to turn backwards and can be removed from the attachment bolt 20.

Moreover, the screw member 11 in the above embodiment was formed in a nut shape by providing a female screw 23 in its core area, but it can also be constructed in a bolt form by providing a male screw 41 in the core area of the screw member 11 as shown in Figure 4. In this figure, 42 is a hexagonal hole and is constructed such that it can be turned by a hexagonal wrench. A detailed explanation of the rest of the construction is omitted in view of the fact that it is the same as the embodiment described above.

By constructing the invention in this way, the rotational tool 22 is secured in pressurized contact under the force of the flange ring 12 in the thrust direction, and this pressurized contact has no energy component in the direction of rotation and, therefore, the screw member 11 can be reliably prevented from being over-tightened due to rotational forces stemming from the reaction to use of the rotational tool 22 in an impactive fashion, and the screw member 11 can be readily removed. Moreover, the impact load of the rotational tool 22 mentioned above is cut off by the torque limiter 18 so that damage to the rotational tool 22 by the impact load can be prevented and, at the same time, damage to the internal parts of the screw 10 for fastening can also be prevented.

Moreover, because the flange ring 12 is pressed under the force of increased torque of the screw feed ring 15, a substantial fastening force and a strong loosening force are obtained at the flange ring 12 for a minor rotational input at the operating ring 13, and the device can be strongly fastened and loosened by a direct manual rotating operation without using a force-enhancing tool.

Moreover, because the reverse screw ring 14 is inserted between the screw member 11 and the screw feed ring 15, the screw feed ring 15 can be more reliably returned to its initial position and a favourable state of use is achieved.

Furthermore, because it comprises a construction with a flange ring 12, operating ring 13, reverse ring 14, screw feed ring 15, holding ring 16 and the like on the outer circumferential area of a screw member 11, this invention can provide a screw for fastening which can be constructed compactly and has good operational properties.

Again, the screw 10 for fastening can be constructed by forming the screw member 11 and the reverse screw ring 14 in the embodiment integrally.

Also, when the fastening load of the screw 10 for fastening is small, the torque limiter 18 shown in the embodiment described above may be omitted and the flange ring 12 may be directly connected to the screw member 11 by spline fitting. In this case, better results are obtained by arranging the construction such that there is somewhat more play in the spline fitting and movement in the axial direction is easy.

WHAT IS CLAIMED IS:

1. Screw for fastening provided with
a screw member with a female screw or a male
screw formed in its core area,
a flange ring which is fitted around the outer
circumferential area of one edge of said screw
member with freedom to turn relative to said screw
member, and the outside surface of which makes
contact with the object to be attached,
an operating ring which is fitted around the
outer circumferential area at the other edge of the
abovementioned screw member with freedom to turn
relative to said screw member, which operating ring
faces the abovementioned flange ring, separated
from it by a prescribed separation, and which
inputs the operational rotational force,
a screw feed ring which is positioned in the
interval between the abovementioned flange ring and
the abovementioned operating ring, which is held
off-center by having part of its female screw area,
which is provided on the relevant inner
circumferential portion, mesh with the male screw
area provided on the outer circumferential area of
the screw member in the said positioning interval,
and which transmits screw feed caused by revolution
to the abovementioned flange ring in the thrust
direction, and
a holding ring fitted around the outer
circumferential area of the abovementioned screw
feed ring, which permits revolution of the said
screw feed ring by holding the said screw feed ring
off-center.
2. Screw for fastening as claimed in claim 1,
in which
a reverse screw ring regulated to a
predetermined screw feed amount is screwed onto the

5 outer circumference of the abovementioned screw member,

 the outer circumference of the said reverse screw ring is provided with a male screw area meshing with part of the female screw area of the
10 abovementioned screw feed ring, and

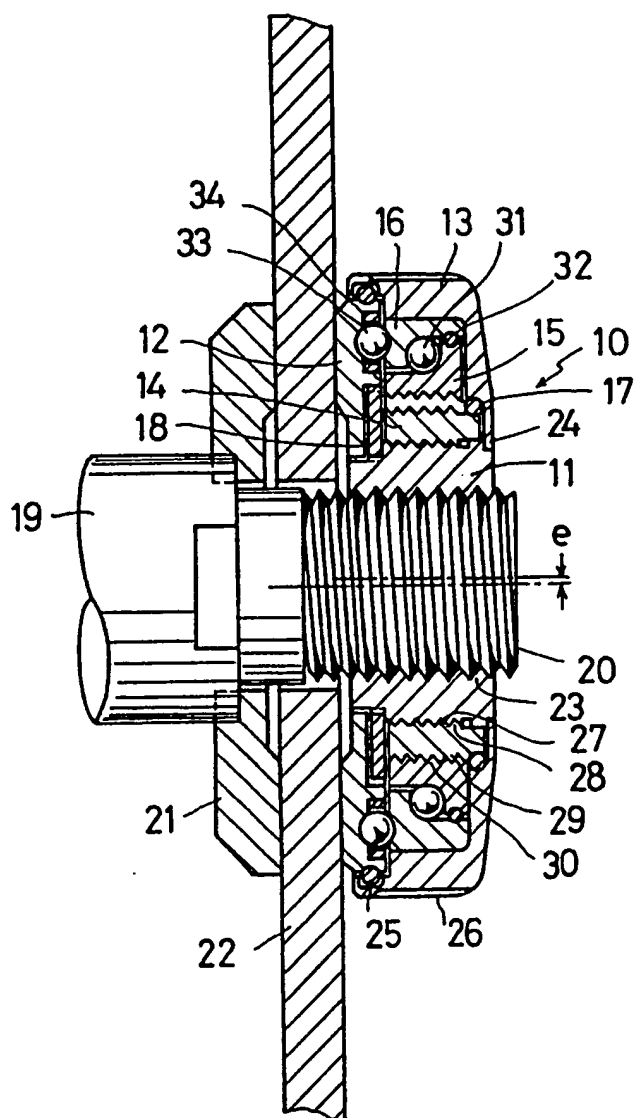
 in which a rotation transmitting member, which transmits the rotation of the operating ring to the reverse screw ring, and which slips at more than a predetermined load, is inserted in the region where
15 the outer circumferential area of the other edge of the abovementioned reverse screw ring and the inner circumferential area of the abovementioned operating ring face each other.

 3. Screw for fastening as claimed in claim 1,

 in which a torque limiter, which cuts off the rotational force from the flange ring at a predetermined load, is inserted between the facing
5 surfaces of the said flange ring and screw member.

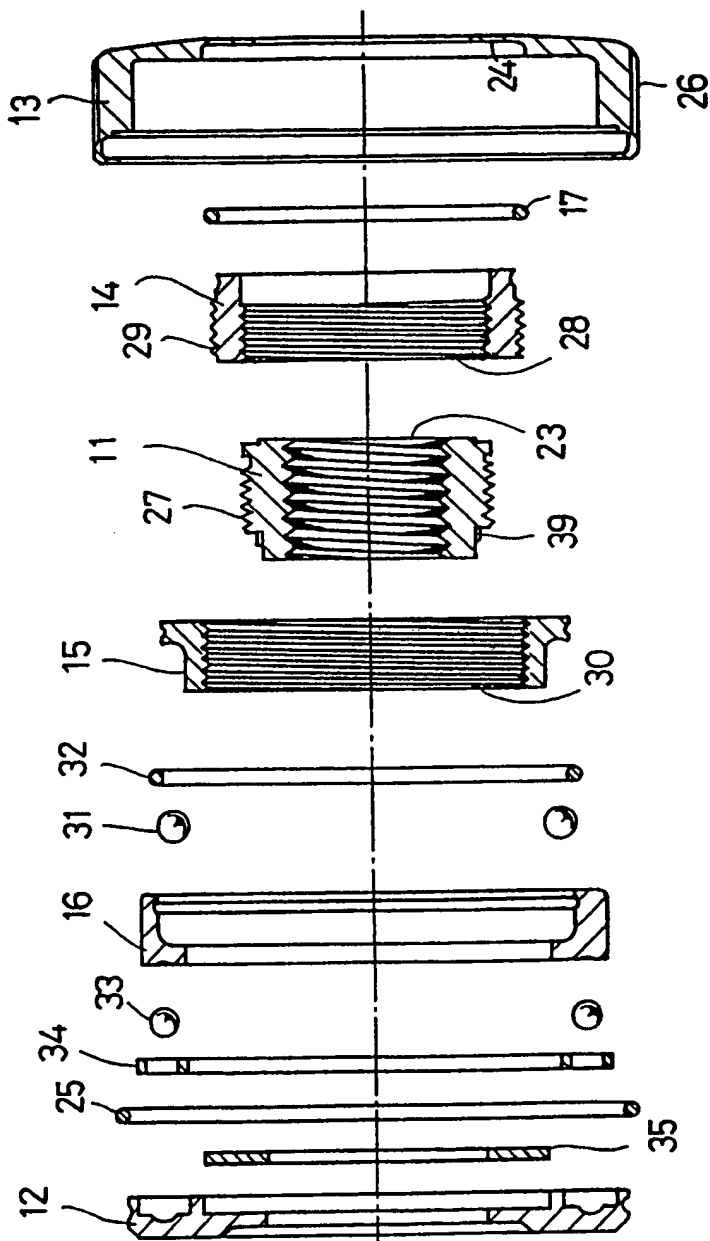
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Fig 1



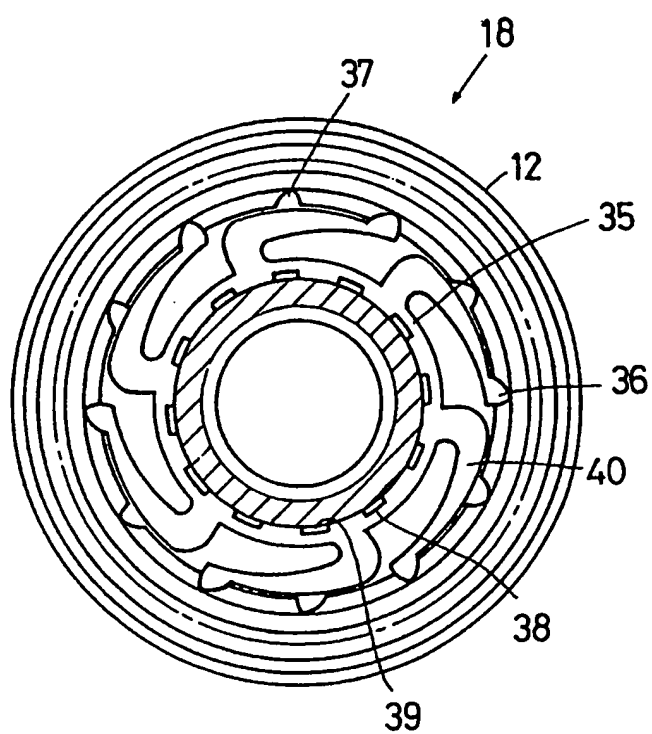
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Fig 2



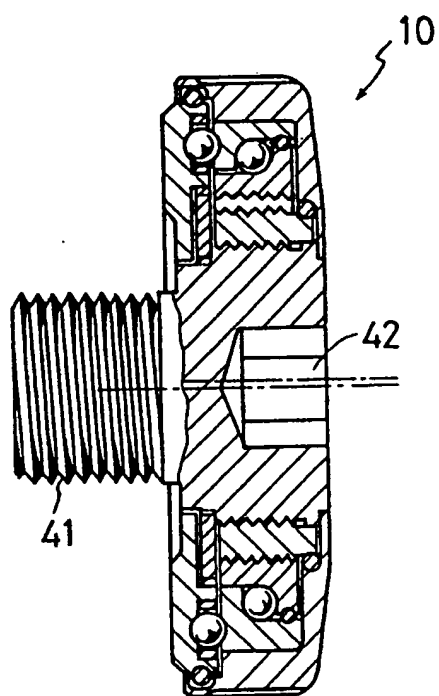
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Fig 3



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Fig 4



A. CLASSIFICATION OF SUBJECT MATTER
 IPC 6 B24B45/00 F16D1/06 F16B33/00

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC 6 B24B F16B F16D B27B

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
Y	WO,A,88 05386 (ROBERT BOSCH GMBH) 28 July 1988 see abstract; figure 1 ---	1
Y	EP,A,0 034 640 (GÄRTNER) 2 September 1981 see page 5, line 7 - page 8, line 19; figures ---	1
A	EP,A,0 497 138 (EATON CORPORATION) 5 August 1992 see the whole document ---	3
A	EP,A,0 480 163 (ROBERT BOSCH GMBH) 15 April 1992 see the whole document -----	1

☐ Further documents are listed in the continuation of box C.

☒ Patent family members are listed in annex.

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